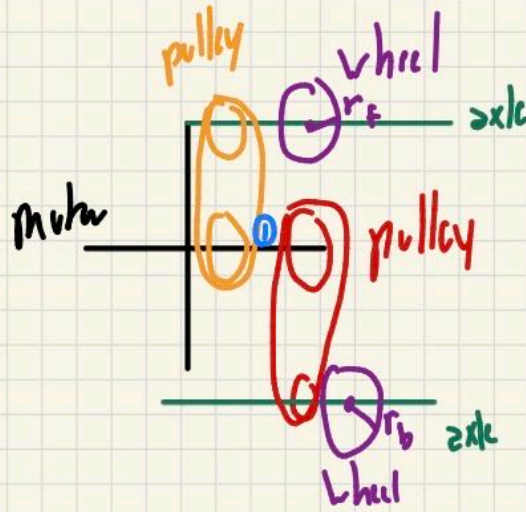


3U16 But Indukt (Rigid) Belt Drive Drive train Analysis

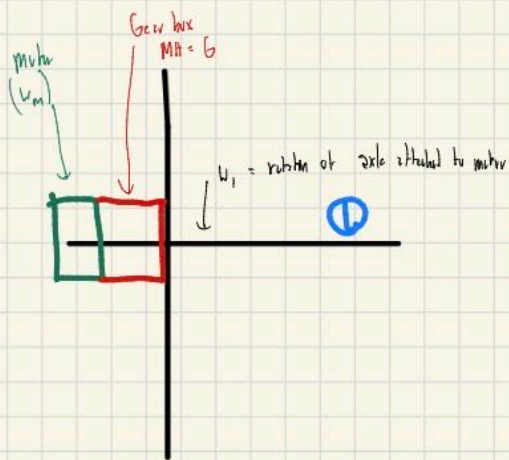
Kedon
Dedabhu

System:

front



Mohr - ①



$$MA = \frac{r_m}{r_{out}} = \frac{r_{out}}{r_m} = "6" > gear$$

$$V_m = V_1$$

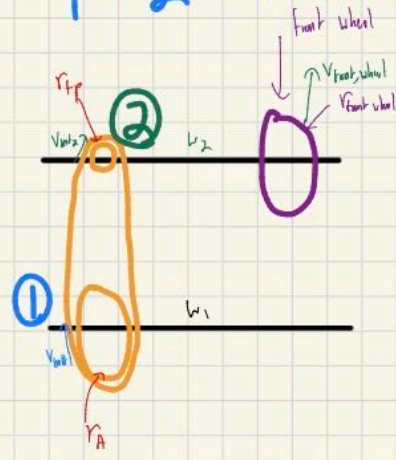
$$r_m \omega_m = r_1 \omega_1$$

$$\omega_1 = \omega_m \frac{r_m}{r_1}$$

$$\omega_1 = \frac{1}{6} \omega_m$$

$$\frac{r_{out}}{r_m} = \frac{r_1}{r_m} = MA$$

1-2 - $V_{front\ wheel}$



$$V_{belt1} = V_{belt2} = V_{belt}$$

$$V_{belt} = \omega_1 r_A$$

$$V_{belt} = \omega_2 r_{fp}$$

$$\omega_2 r_{fp} = \omega_1 r_A$$

$$\omega_2 = \frac{r_A}{r_{fp}} \omega_1$$

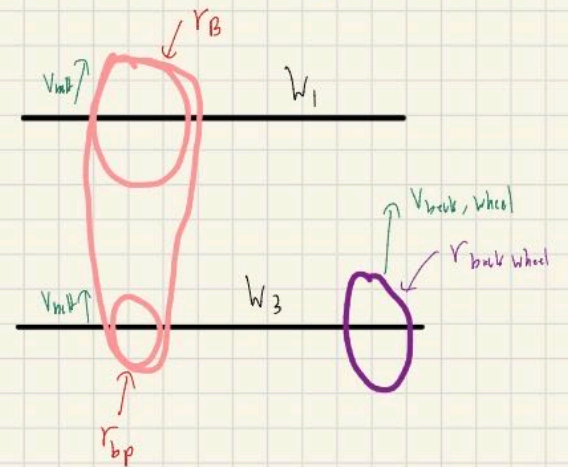
$$\omega_2 = \frac{1}{6} \frac{r_A}{r_{fp}} \omega_m$$

$$\rightarrow \frac{r_A}{r_{fp}} = \frac{\text{Teeth Motor Pulley (Driver)}}{\text{Teeth Front Pulley (Driven)}} = \frac{1}{\text{Front Speed Ratio}} = \frac{1}{R_f}$$

$$\omega_2 = \frac{1}{6 R_f} \omega_m$$

$$V_{front\ wheel} = \omega_2 r_{front\ wheel} = \frac{r_{front\ wheel}}{6 R_f} \omega_m$$

1-3 - $V_{back\ wheel}$



$$V_{belt} = r_B \omega_1 = r_{bp} \omega_3$$

$$\omega_3 = \frac{r_B}{r_{bp}} \omega_1$$

$$\omega_3 = \frac{r_B}{r_{bp}} \left(\frac{1}{6} \omega_m \right)$$

$$\rightarrow \frac{r_B}{r_{bp}} = \frac{\text{Teeth Motor Pulley (Driver)}}{\text{Teeth Back Pulley (Driven)}} = \frac{1}{R_{back}}$$

$$\omega_3 = \frac{1}{6 R_{back}} \omega_m$$

$$V_{back\ wheel} = r_{back\ wheel} \omega_3$$

$$V_{back\ wheel} = \frac{r_{back\ wheel}}{6 R_{back}} \omega_m$$

So...

* We need $V_{\text{back wheel}} = V_{\text{front wheel}}$ in order to go straight... obviously

→ So:

$$\frac{r_{\text{back wheel}}}{R_{\text{back}}} \cdot \frac{\omega_m}{G} = \frac{\omega_m}{G} \cdot \frac{r_{\text{front wheel}}}{R_{\text{front}}}$$

$$\frac{r_{\text{back wheel}}}{R_{\text{back}}} = \frac{r_{\text{front wheel}}}{R_{\text{front}}}$$

$$* R_{\text{back}} = \frac{\text{Teeth Back Pulley}}{\text{Teeth of corresponding Motor Pulley}}$$

$$* R_{\text{front}} = \frac{\text{Teeth Front Pulley}}{\text{Teeth of corresponding Motor Pulley}}$$

Assuming this is satisfied:

We can say:

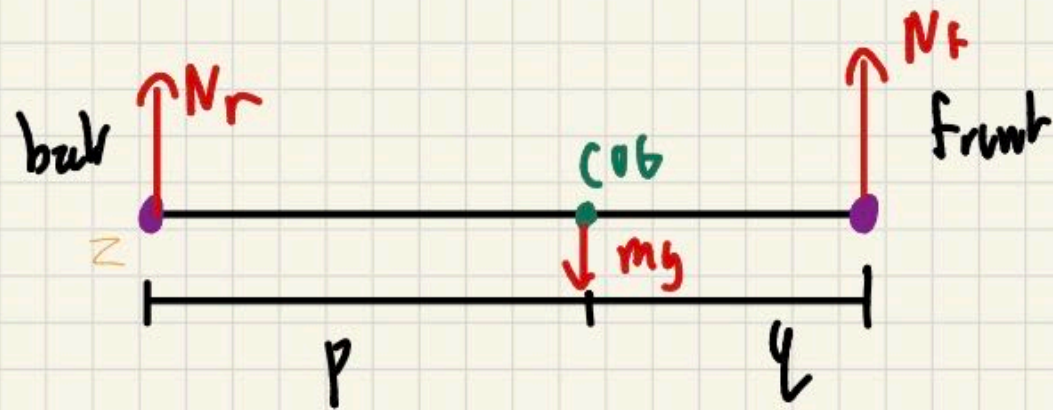
$$\bullet V_{\text{front}} = V_{\text{back}} = V$$

$$\bullet V = \frac{r_{\text{front wheel}}}{G R_f} \omega_m = \frac{r_{\text{back wheel}}}{G R_b} \omega_m$$

$$\rightarrow \frac{r_{\text{front wheel}}}{G R_f} = \frac{r_{\text{back wheel}}}{G R_b} \equiv k_{\text{sys}}$$

$$* V = k_{\text{sys}} \omega_m$$

Status of System:



$$M_z = 0 = N_f (pq) - mg(p)$$

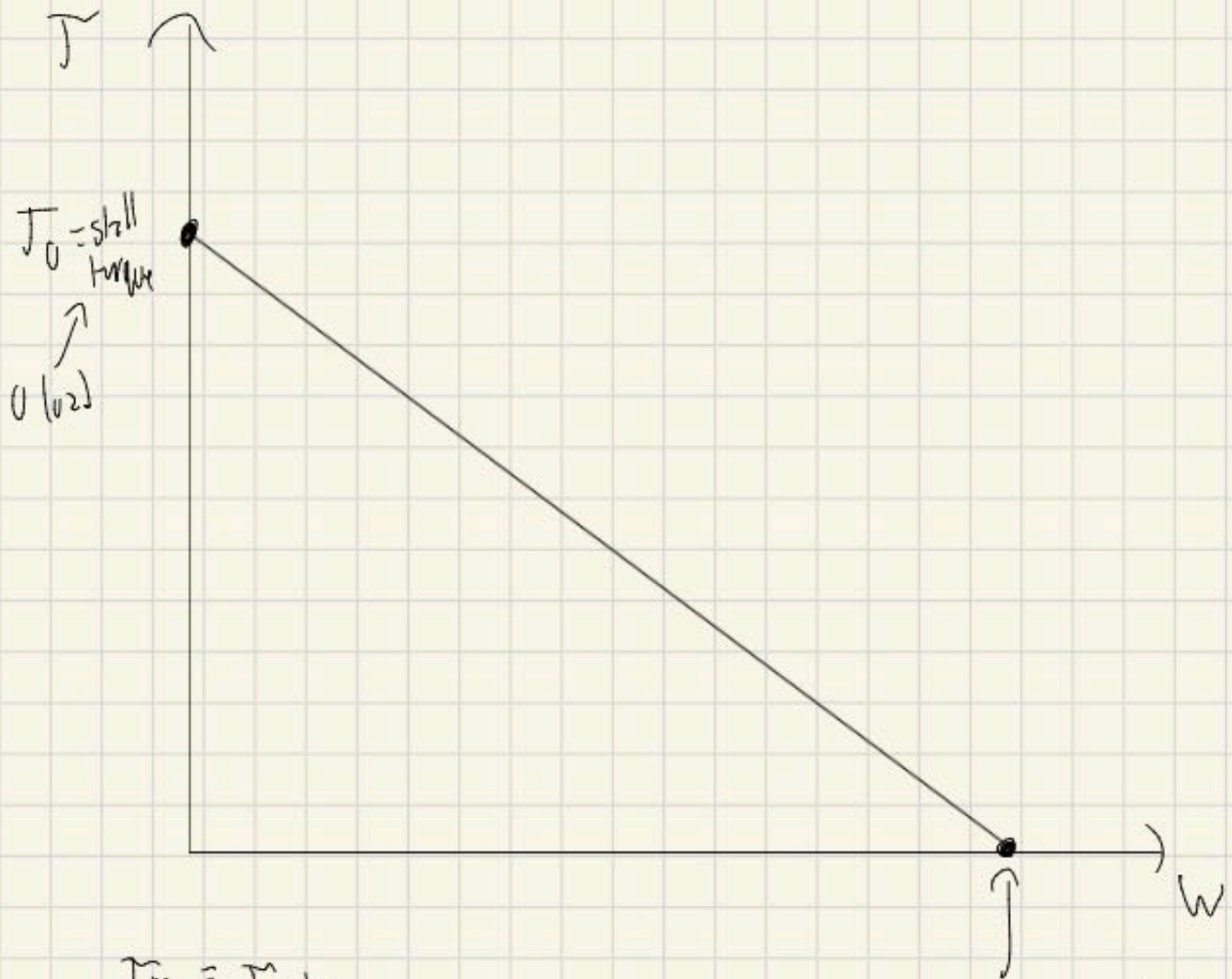
$$N_f = \frac{mgp}{pq} = mg \left(\frac{p}{pq} \right)$$

$$\sum F_y = 0 \Rightarrow N_f - mg + N_b = 0$$

$$N_b = mg \left(1 - \frac{p}{pq} \right)$$

* Where $mg = \text{Weight of 1 side} = \frac{\text{Total Weight}}{2}$

DC Motor T - ω Curve



$$T_m \equiv T_{\text{motor}}$$

$$\omega_m \equiv \omega_{\text{motor}}$$

$$\omega_{NL} = \text{no load}$$

$T_{\text{put}} \equiv T_m = -\frac{T_0}{\omega_{NL}} \omega_m \oplus T_0 = T_0 \left(1 - \frac{\omega_m}{\omega_{NL}} \right)$

just renaming for later purposes

$$T_0 = I_{\text{max}} \cdot k_t$$

motor properties

$$\omega_{NL} = \text{Voltage} \cdot k_v$$

motor properties

Work Energy Conservation

Power in = Power out

Power = $\tau \cdot \omega = \frac{d}{dt} (kE)_{\text{total}}$ in some cases = Fv

$$T_m \omega_m = \frac{d}{dt} \left(kE_{\text{chassis}} \oplus kE_{\text{hub pulley}} \oplus kE_{\text{wheel pulley}} \oplus kE_{\text{motor shaft}} \right)$$

$$T_m \omega_m = \frac{d}{dt} \left(\frac{1}{2} m v^2 \oplus \frac{1}{2} I_{fp} \omega_f^2 \oplus \frac{1}{2} I_{rp} \omega_r^2 \oplus \frac{1}{2} I_m \omega_m^2 \right)$$

$v = \omega_{sys} r$ $\omega_f = \frac{\omega_m}{R_f}$ $\omega_r = \frac{\omega_m}{R_r}$

$$T_m \omega_m = \frac{d}{dt} \left(\frac{1}{2} m k_{sys}^2 \omega_m^2 \oplus \frac{1}{2} \frac{I_{fp}}{R_f^2} \omega_m^2 \oplus \frac{1}{2} \frac{I_{rp}}{R_r^2} \omega_m^2 \oplus \frac{1}{2} I_m \omega_m^2 \right)$$

$$T_m \omega_m = \frac{1}{2} \left[m k_{sys}^2 \oplus \frac{I_{fp}}{R_f^2} \oplus \frac{I_{rp}}{R_r^2} \oplus I_m \right] \frac{d}{dt} (\omega_m^2)$$

I_{eq}

$$T_m \omega_m = I_{eq} \omega_m \alpha_m$$

$$T_m = I_{eq} \alpha_m$$

$$\rightarrow \alpha = \alpha_m k_{sys}$$

$$\text{So } T_m = \frac{I_{eq} \alpha}{k_{sys}}$$

$$\alpha = \frac{T_m k_{sys}}{I_{eq}}$$

~~X~~

However T_m might need to be limited bc of traction

$$\text{Max Traction} = \text{Traction front} @ \text{Traction back}$$

* Since our system is 4WD (rigid connections)

→ Torque will go to where there is traction

→ If one has more traction than the other:

→ The other will stay at its traction limit and act as a free axle... the other wheel will pull it up in speed

↑
It won't slip bc no torque above its max is applied!

(A advantage of 4WD)

Conclusions:

$$\text{Power out} = \frac{d}{dt} \left(\frac{1}{2} k E \right) \approx \text{Power in}$$

$$T_m W_m = F_F v \oplus F_r v$$

$$\cancel{T_m W_m} = (F_F \oplus F_r) \cancel{W_m} b_{\text{sys}}$$

$$T_{\text{limit}} = T_m = M (N_f \oplus N_r) b_{\text{sys}}$$

$$\times N_f \oplus N_r = m g = \frac{\text{total weight}}{2}$$

So

$$T_m \begin{cases} T_{\text{potential}} & \text{if } T_{\text{potential}} < T_{\text{limit}} \\ T_{\text{limit}} & \text{if } T_{\text{potential}} \geq T_{\text{limit}} \end{cases}$$

So

$$\alpha_m = \frac{T_m}{I_{\text{eq}}} > \text{a function of time}$$

$$W_m = \int \alpha_m dt$$

$$V = b_{\text{sys}} W_m = b_{\text{sys}} \int \alpha_m dt$$